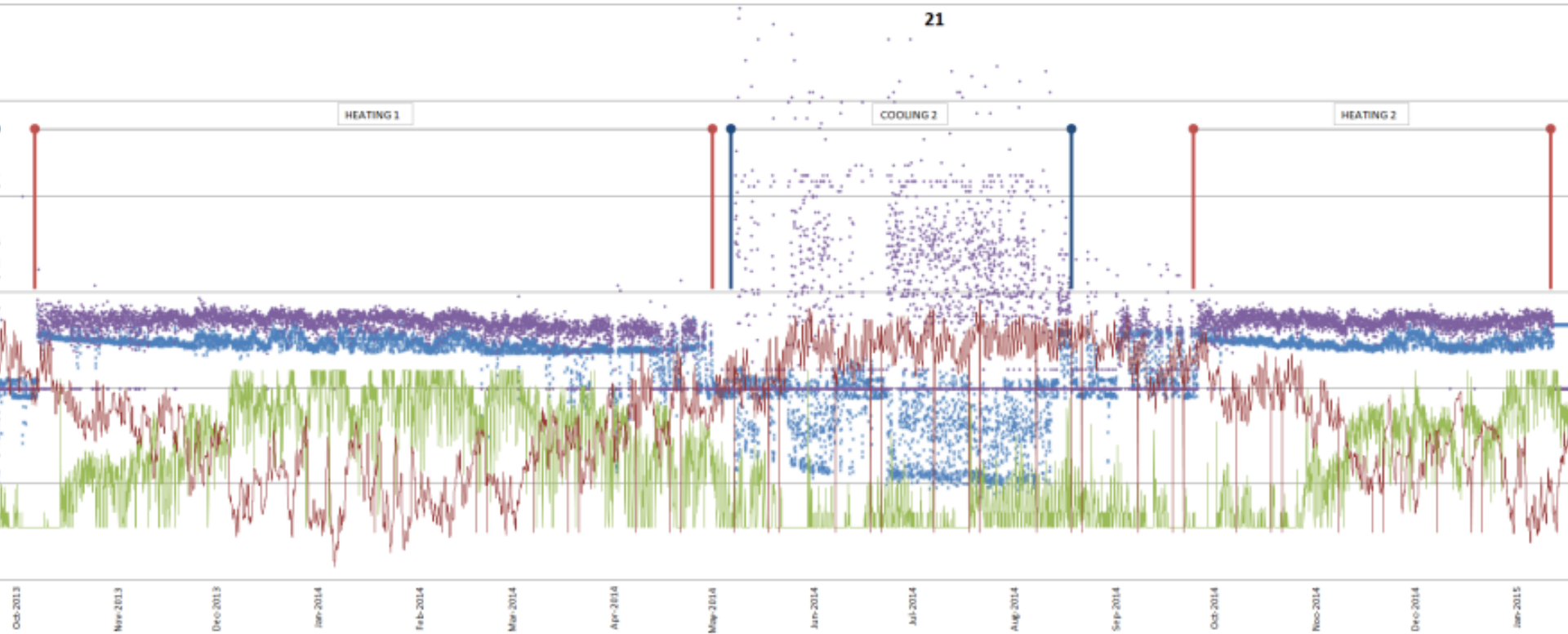


GSHPs and ASHPs

Real Performance Results in Cold Climates



University of Minnesota Cold Climate Housing Program
Center for Sustainable Building Research (CSBR)
Center for Energy and Environment (CEE)

GSHPs and ASHPs - Real Performance Results in Cold Climates

Outline

- 1) Ground source heat pump monitoring project
 - i. Background
 - ii. Monitoring
 - iii. Modeling

- 2) Air source heat pump monitoring project
 - i. Background
 - ii. Monitoring Results

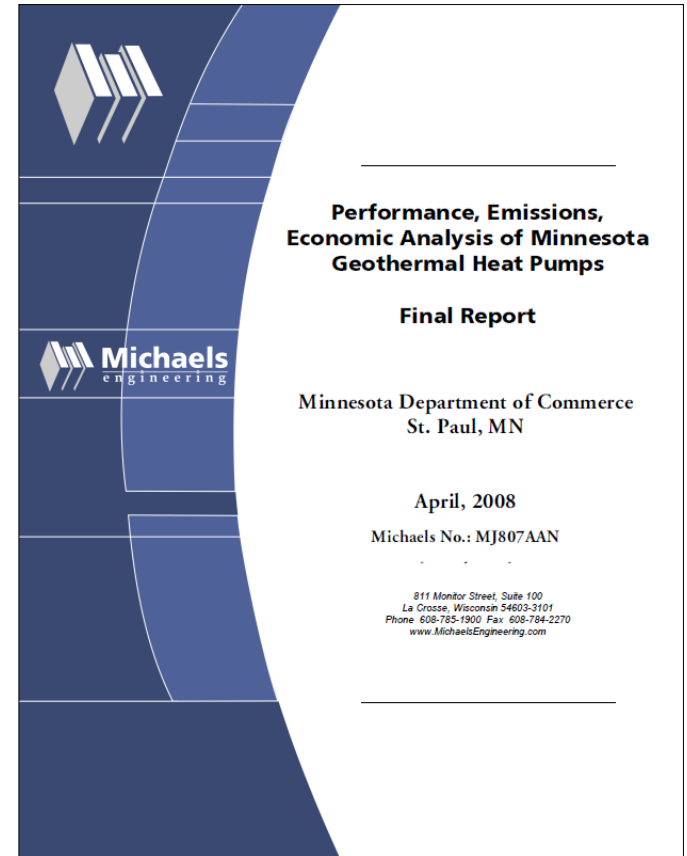
- 3) Discussion, comparison, & implications for Passive House

GSHP - background

Prior research:

Michaels Engineering Report for MN
Department of Commerce (2008)

- 1) Found negative life cycle savings and increased CO2 emissions for residential installations.
- 2) Widely criticized by the GSHP industry for using modeled energy results **based on assumed GSHP efficiency levels.**
- 3) Assumed 3.3 heating COP, 4.1 cooling COP for the energy modeling study

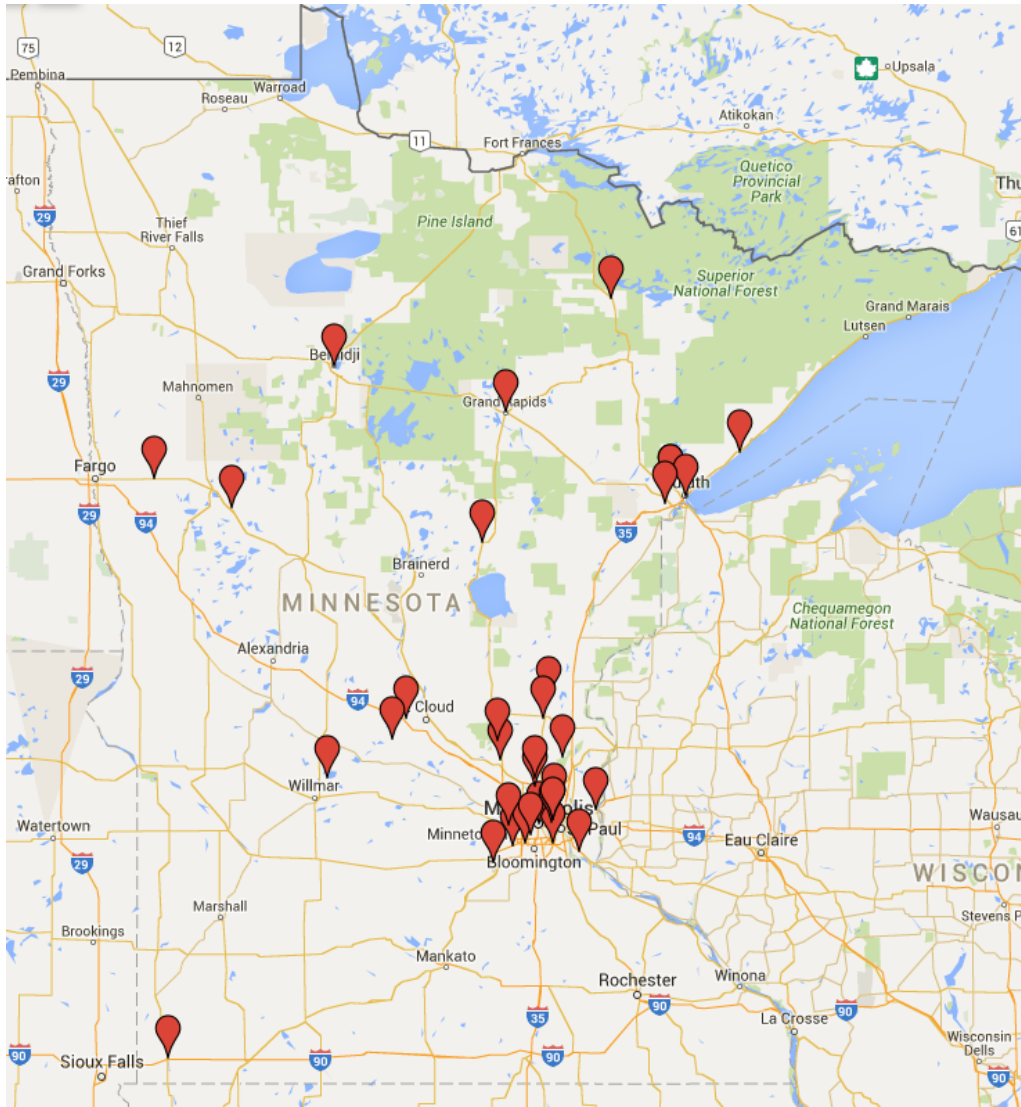


GSHP - background

University of Minnesota's Cold Climate Housing Program proposed revised research approach:

- 1) Multi-year in-situ monitoring of GSHP systems
- 2) Energy modeling comparison to standard GFA systems using COP results from monitoring
- 3) Broader environmental and policy assessment of GSHP technology, including an LCA study

GSHP - monitoring

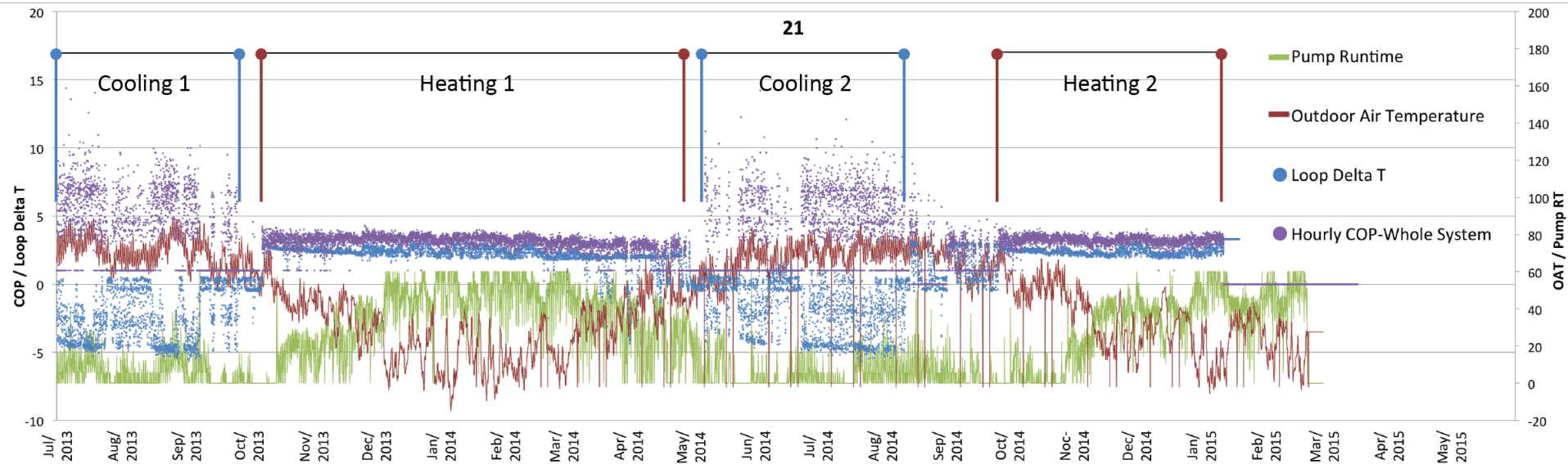


Original goal:
50 monitored systems in
various climate zones

After drop-outs,
equipment failures and
data issues, 37 systems
were used for the report.

Monitoring system design and
installation provided by Mike LeBeau,
CR Building Performance Specialists.

GSHP - monitoring

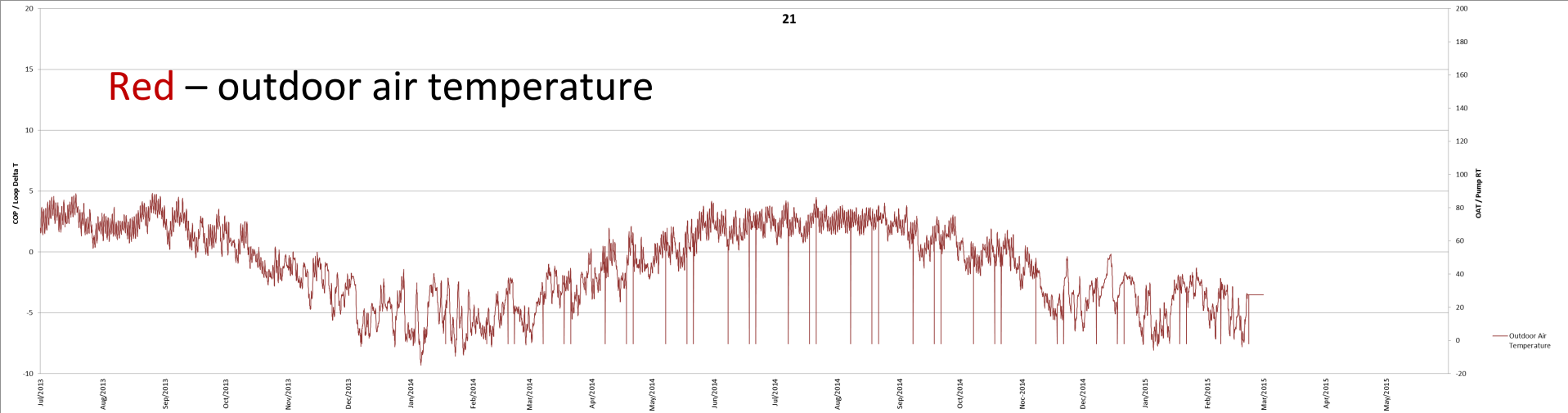


4-variable graphs used to identify heating and cooling seasons

GSHP - monitoring

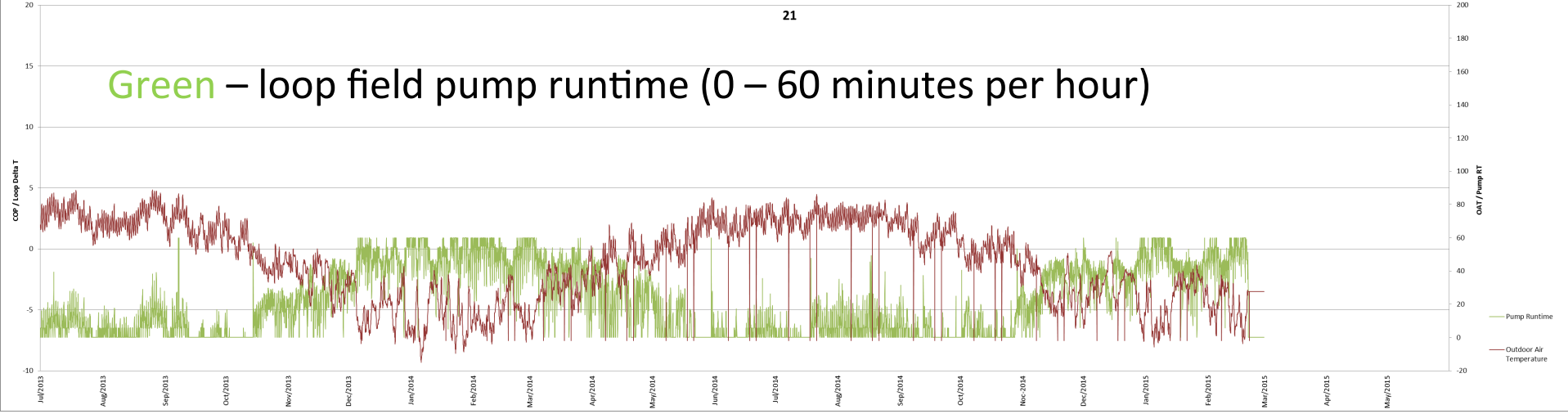
21

Red – outdoor air temperature



21

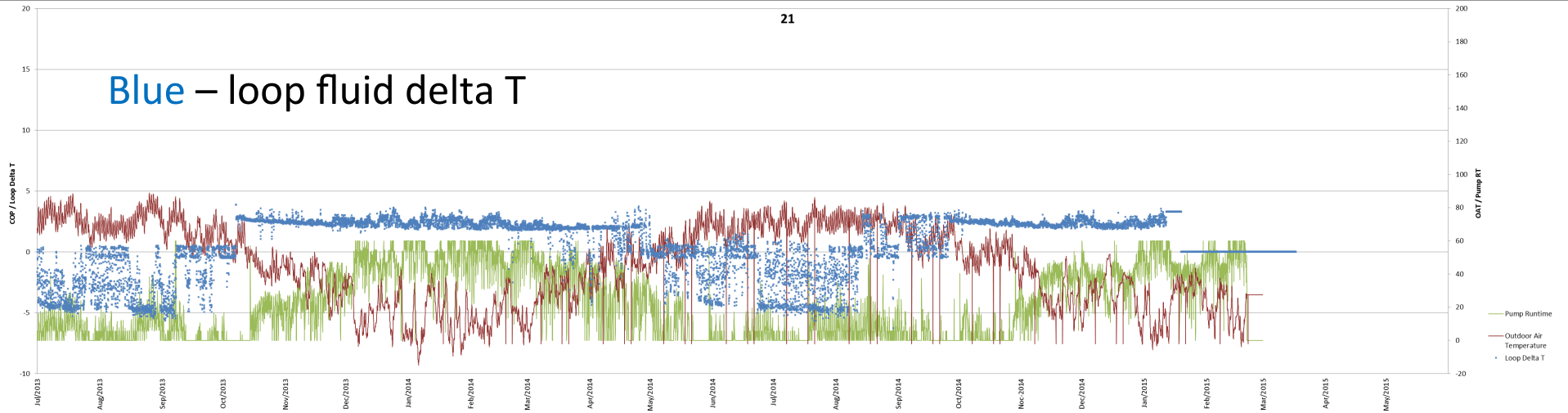
Green – loop field pump runtime (0 – 60 minutes per hour)



GSHP - monitoring

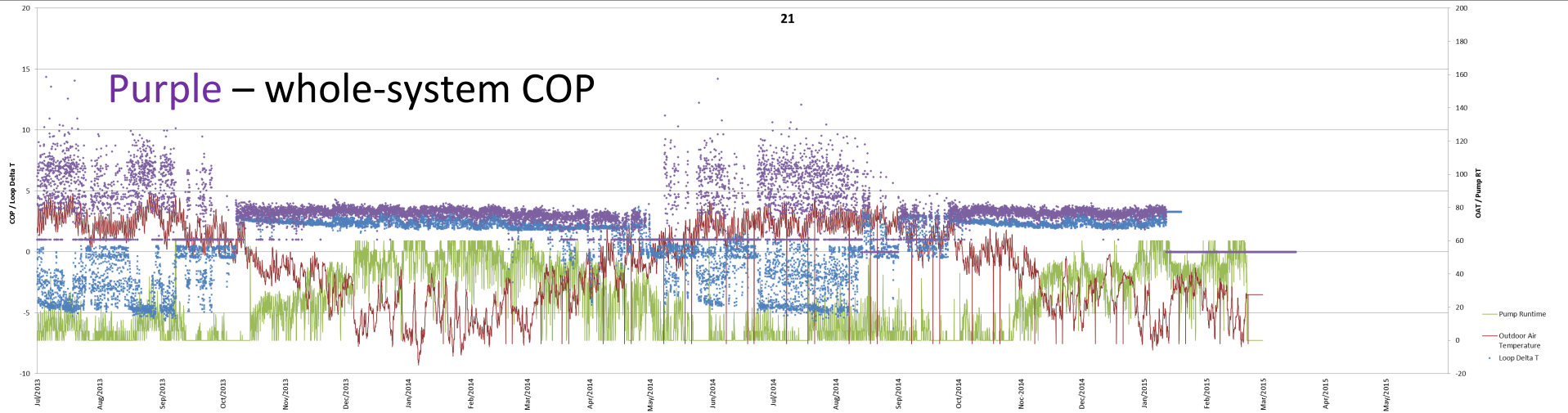
21

Blue – loop fluid delta T

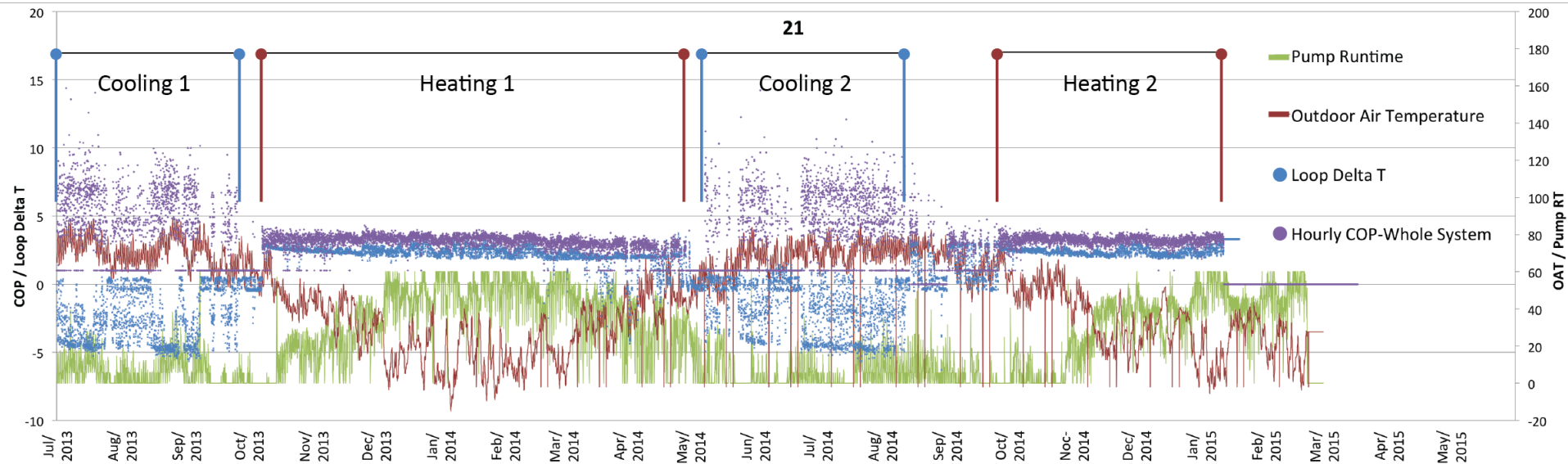


21

Purple – whole-system COP



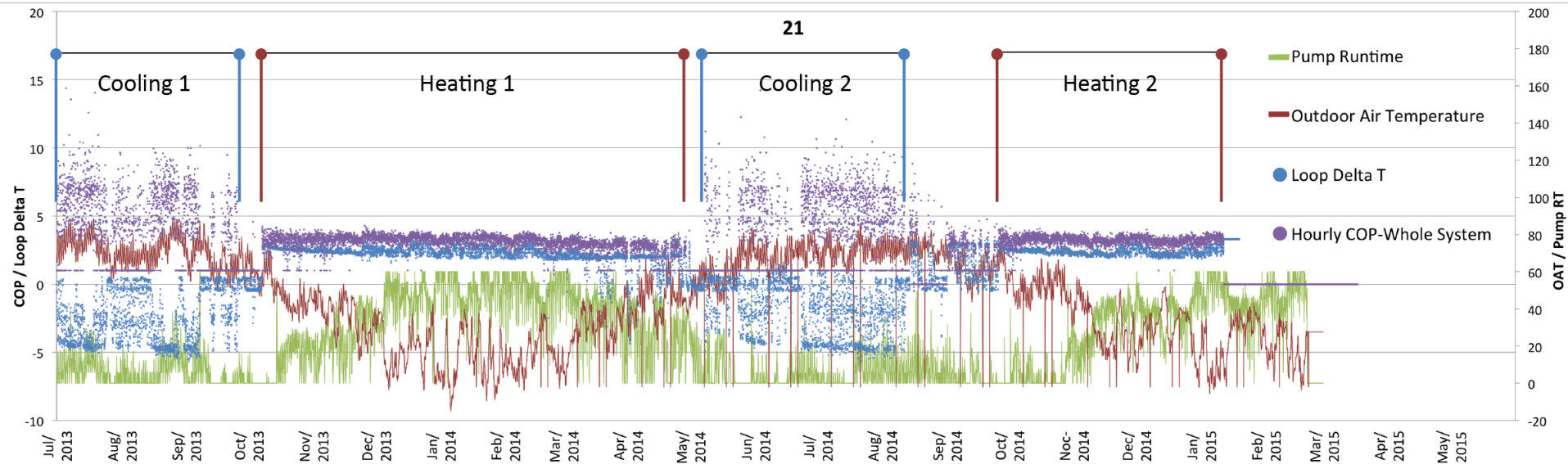
GSHP - monitoring



$$\text{COP (heating)} = (\text{Energy Delivered} + \text{Energy Consumed}) / \text{Energy Consumed}$$

$$\text{COP (cooling)} = (\text{Energy Extracted} - \text{Energy Consumed}) / \text{Energy Consumed}$$

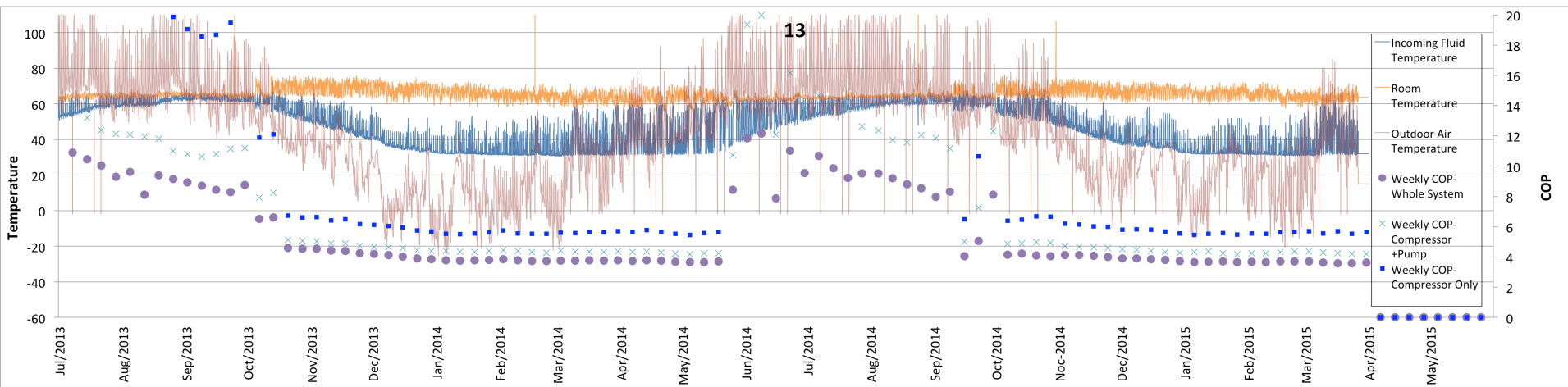
GSHP - monitoring



Take-aways:

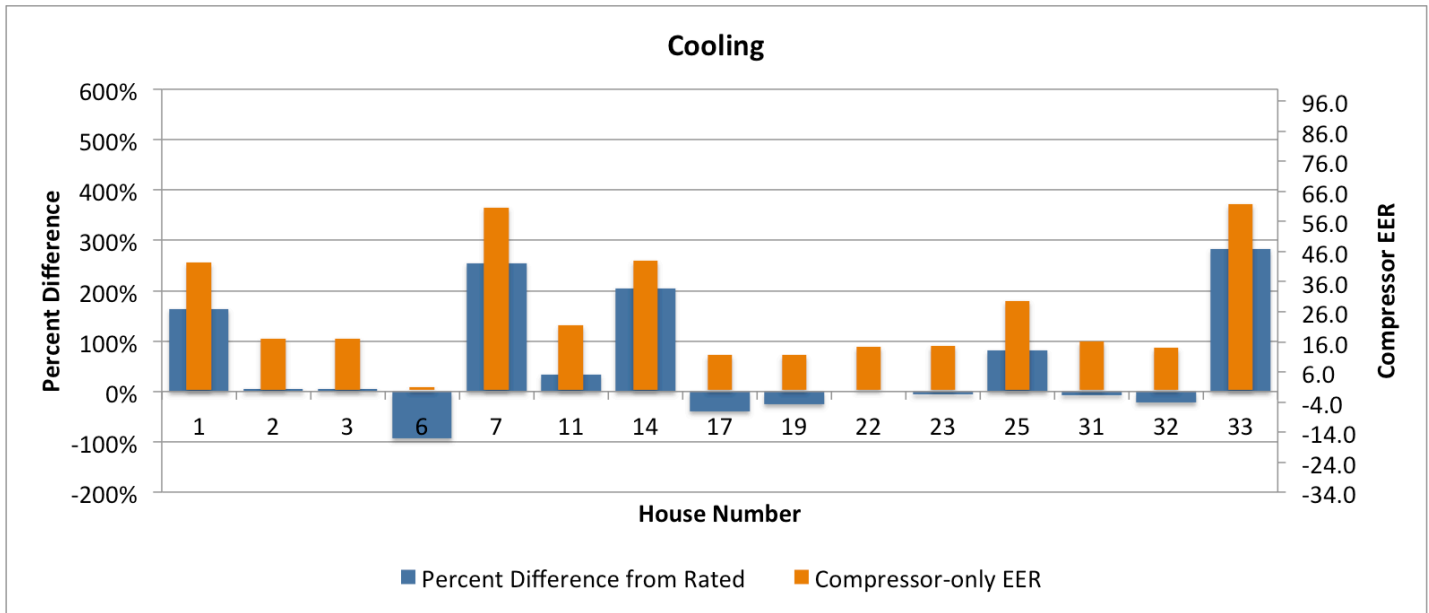
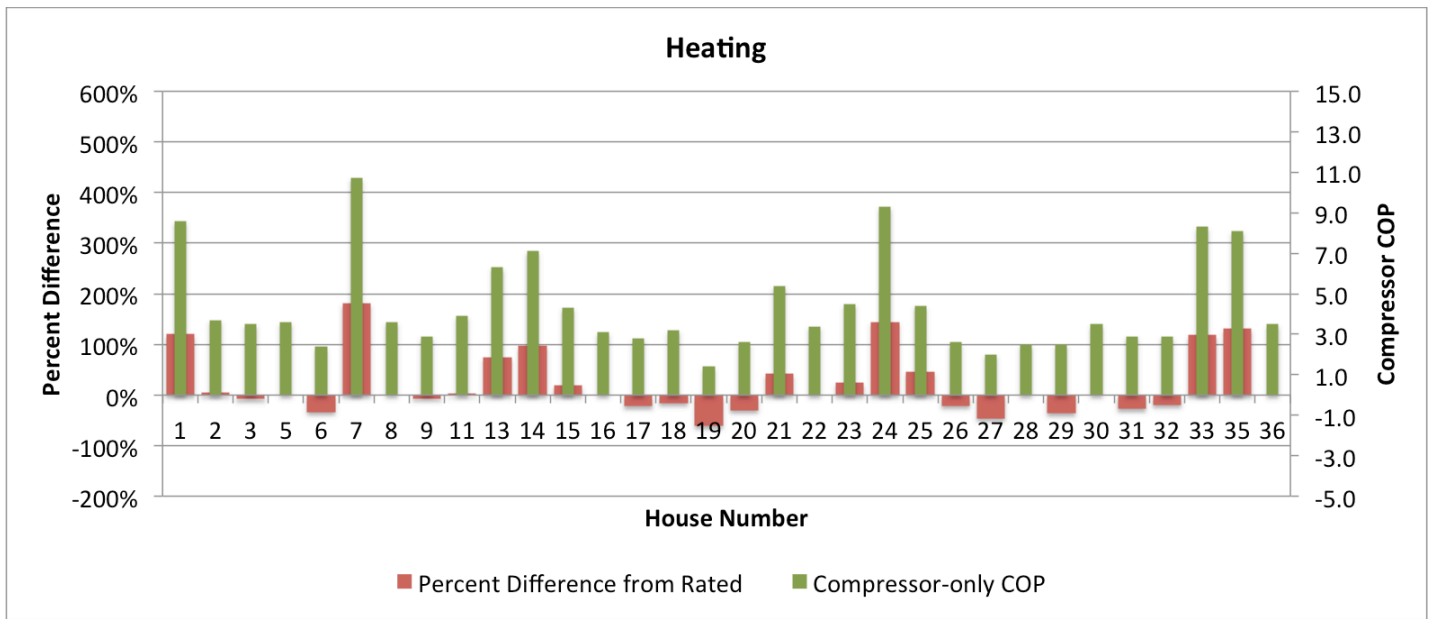
1. Cooling season is short and run time is highly irregular
2. Greater loop fluid delta T corresponds to higher COPs
3. This system is probably undersized (as are most GSHPs to save install cost)

GSHP - monitoring



Take-aways:

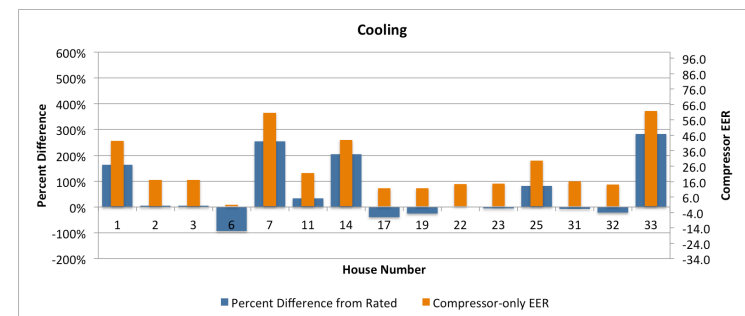
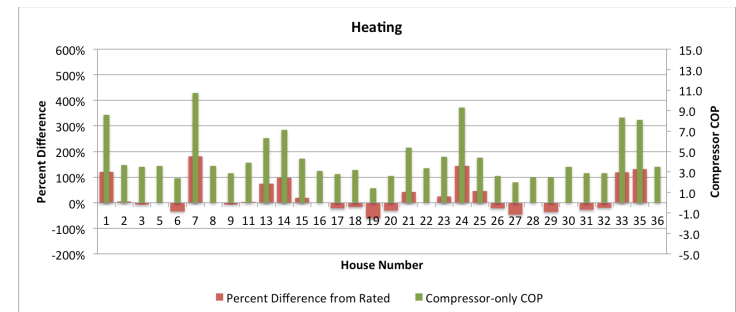
1. Noticeable decline in COP during the course of a heating or cooling season
 - i. Heating whole system COP – 4.5 down to 3.8
 - ii. Cooling whole system COP – 10 down to 8
2. Addition of loop field pump energy drops heating COP by close to 2 (roughly 6 to 4, for this high-performing system)
3. Loop field temperature bottoming out near 32F, peaking at 60F



GSHP - monitoring

Large variability in compressor performance, especially on cooling side.

- i. Variation from each other
- ii. Variation from expected

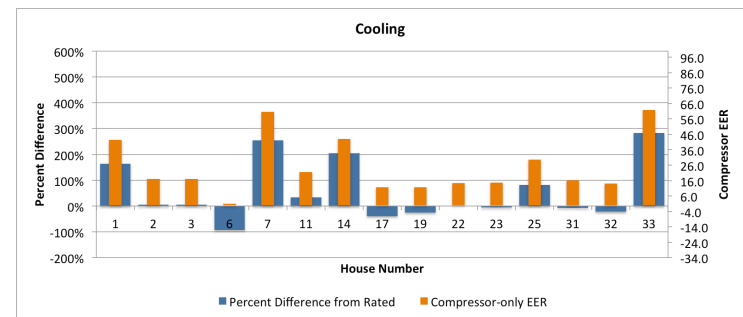
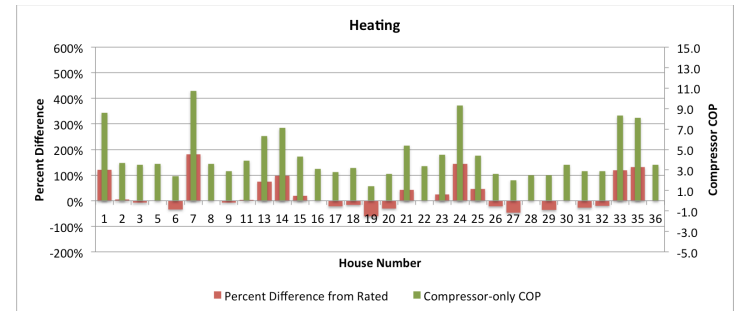


GSHP - monitoring

Large variability in compressor performance, especially on cooling side.

- i. Variation from each other
- ii. Variation from expected

Is the homeowner going to get what they paid for???

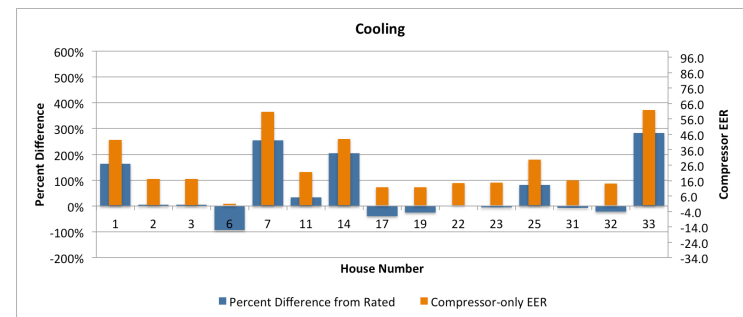
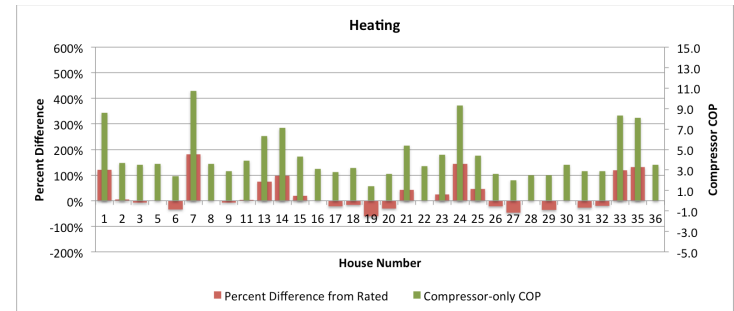


GSHP - monitoring

Large variability in compressor performance, especially on cooling side.

- i. Variation from each other
- ii. Variation from expected

Welcome to the high-stakes table!



GSHP - monitoring

Final measured heating and cooling COPs (compressor + loop pump)

	Heating COP	Cooling COP
Minimum Value	1.51	0.23
25th Percentile	2.88	3.91
Median	3.19	5.20
75th Percentile	3.75	10.13
Maximum Value	7.19	23.44

GSHP - monitoring

Final measured heating and cooling COPs (compressor + loop pump)

	Heating COP	Cooling COP
Minimum Value	1.51	0.23
“low” COP	2.88	3.91
“mid” COP	3.19	5.20
“high” COP	3.75	10.13
Maximum Value	7.19	23.44

Values used for energy modeling

Michaels Engineering report assumptions: heating COP 3.3, cooling COP 4.1

GSHP - modeling

9 base cases used to determine GSHP savings in a wide variety of possible installation scenarios

		Home Energy Efficiency		
		Low	Medium	High
		Pre-energy code	1980's MN energy code	2015 MN energy code
Home Size	2000 sf (small)	1	4	7
	3000 sf (medium)	2	5	8
	4000 sf (large)	3	6	9

GSHP - modeling

9 cases x 3 efficiency levels = 27 GSHP models

+

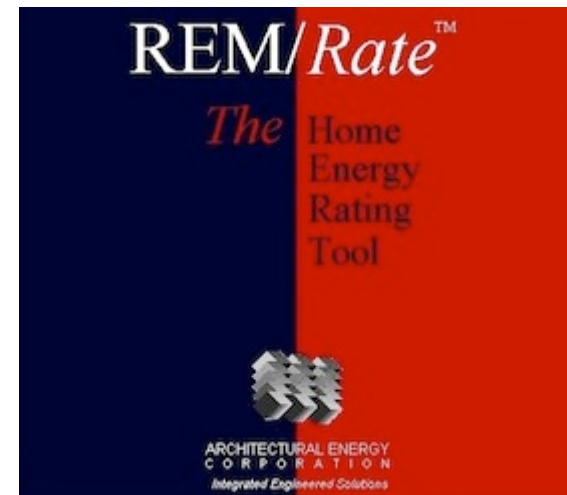
9 base cases w fuel-fired GFA

+

9 GSHP models w desuperheaters

=

45 models total



GSHP - modeling

Complications:

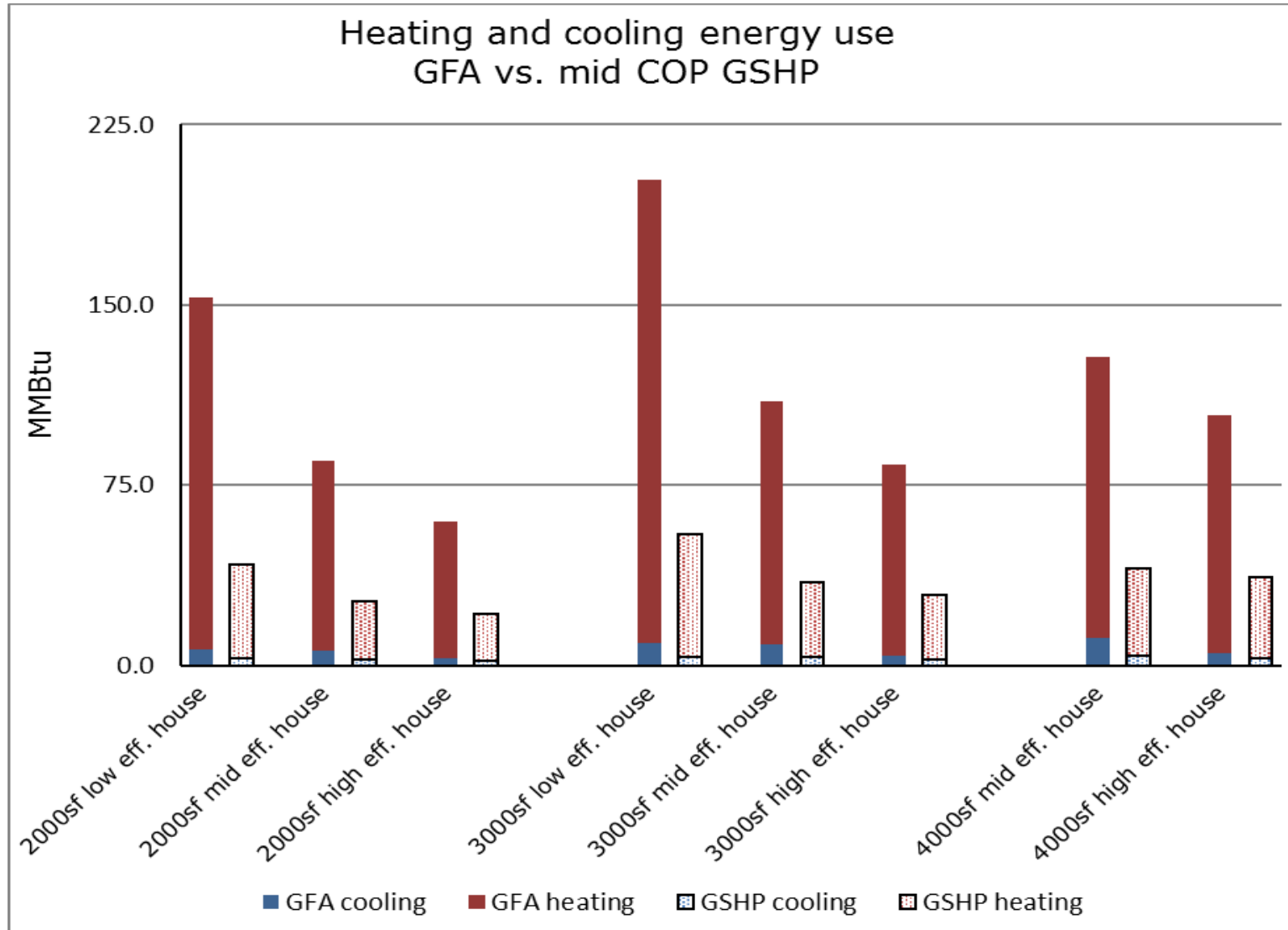
- 1) Different supply air temperatures meant that fan airflows and therefore fan energy would be different for each system at a given heat load. And REM Rate does not do a good job with fan energy.
- 2) GFA systems are oversized, whereas GSHP systems rely on backup heat systems – How much backup? How much oversizing?
- 3) GSHP systems can provide hot water savings with the use of a desuperheater

GSHP - modeling

Fan airflows and energy use based on supply air temperature

Heating equipment	Assumed Supply Air Temp	Source	Resulting fan energy factor
70 AFUE GFA	140°F	CEE, PARR (upper end of range)	0.86
80 AFUE GFA	130°F	CEE, PARR (lower end of range)	1.0 (base case)
90 AFUE GFA	115°F	CEE, PARR (lower end of range)	1.32
GSHP (high, med, low COPs)	95°F	Field data, average heating supply air temp	2.3

GSHP - modeling

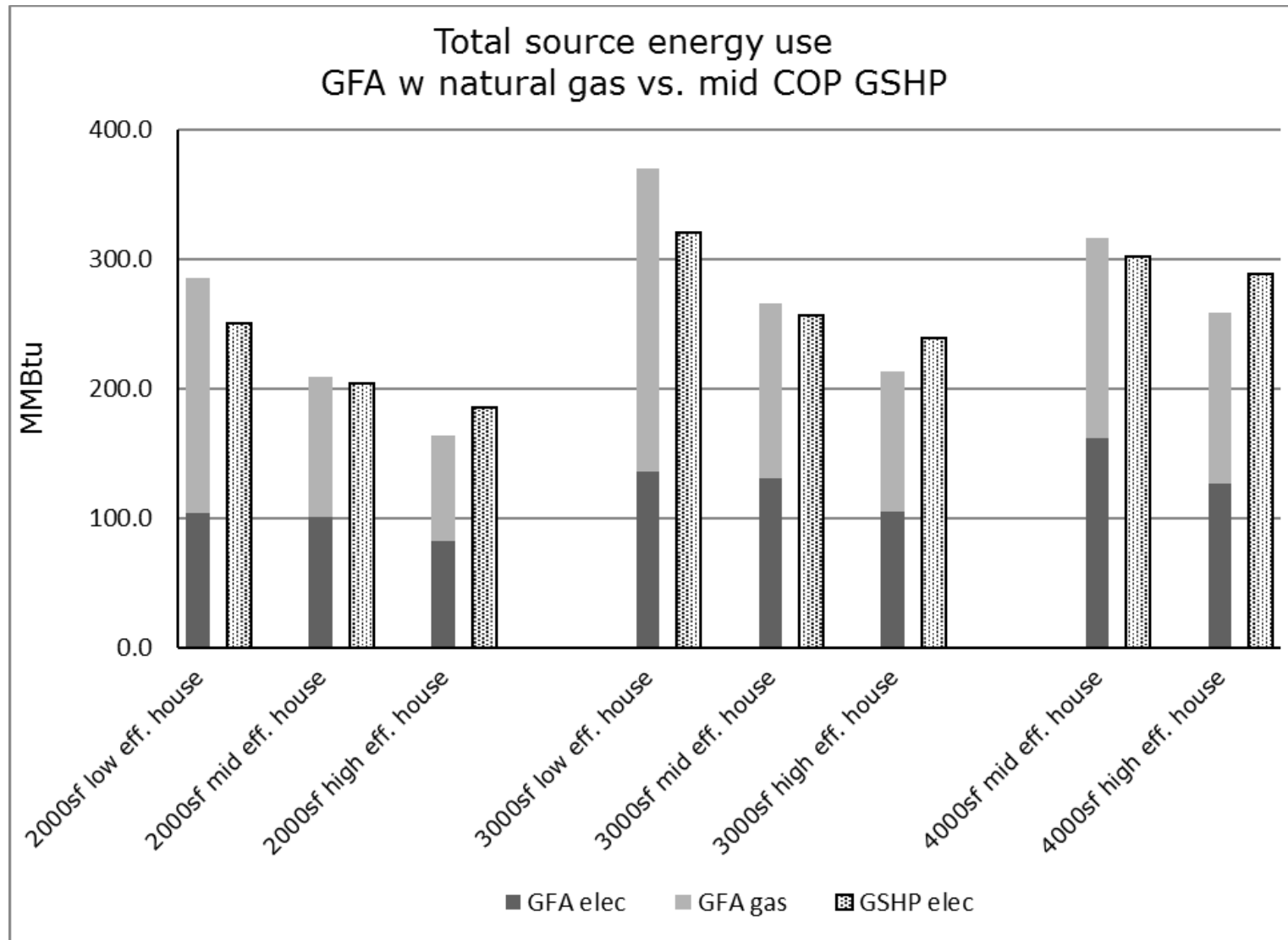


GSHP - modeling

Translate site to source energy...

Source energy factors		referenced source
grid electricity	3.15	ANSI Std. 105-2014 (national avg.)
natural gas burned in combustion appliance	1.09	ANSI Std. 105-2014 (national avg.)
LP burned in combustion appliance	1.15	ANSI Std. 105-2014 (national avg.)

GSHP - modeling

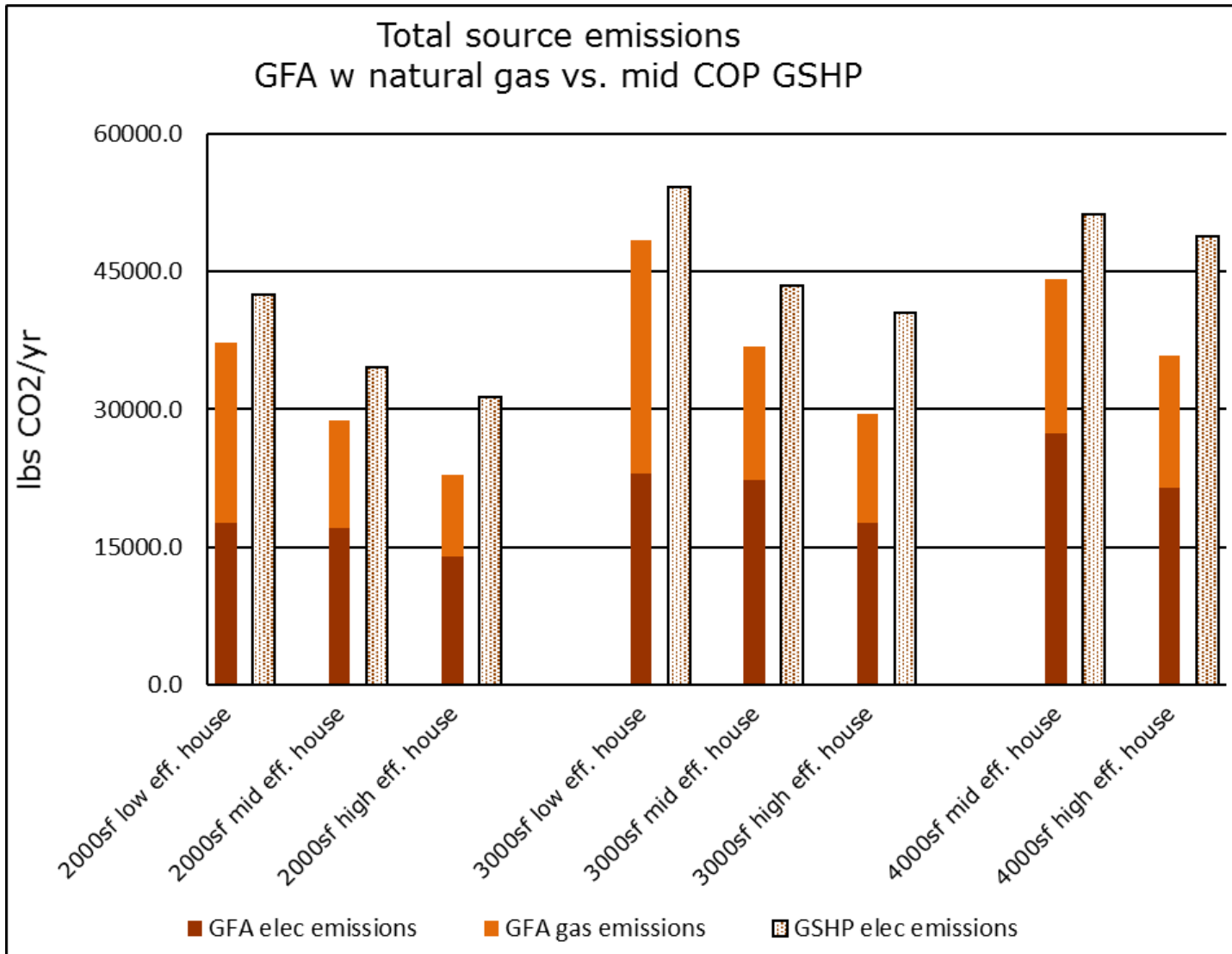


GSHP - modeling

Translate site energy to CO2 emissions...

Energy Type	Rate	Units	Source	lbs CO2/MMBtu
electricity	1.82	lbs CO2/kWh	MN PCA	533.4
natural gas	11.79	lbs CO2/therm	US EIA Form EIA-1605 Instructions "Voluntary Reporting of Greenhouse Gases", 2010	117.9
propane	12.55	lbs CO2/gallon	US EIA Form EIA-1605 Instructions "Voluntary Reporting of Greenhouse Gases", 2010	137.4

GSHP - modeling



GSHP - modeling

Translate to energy costs...

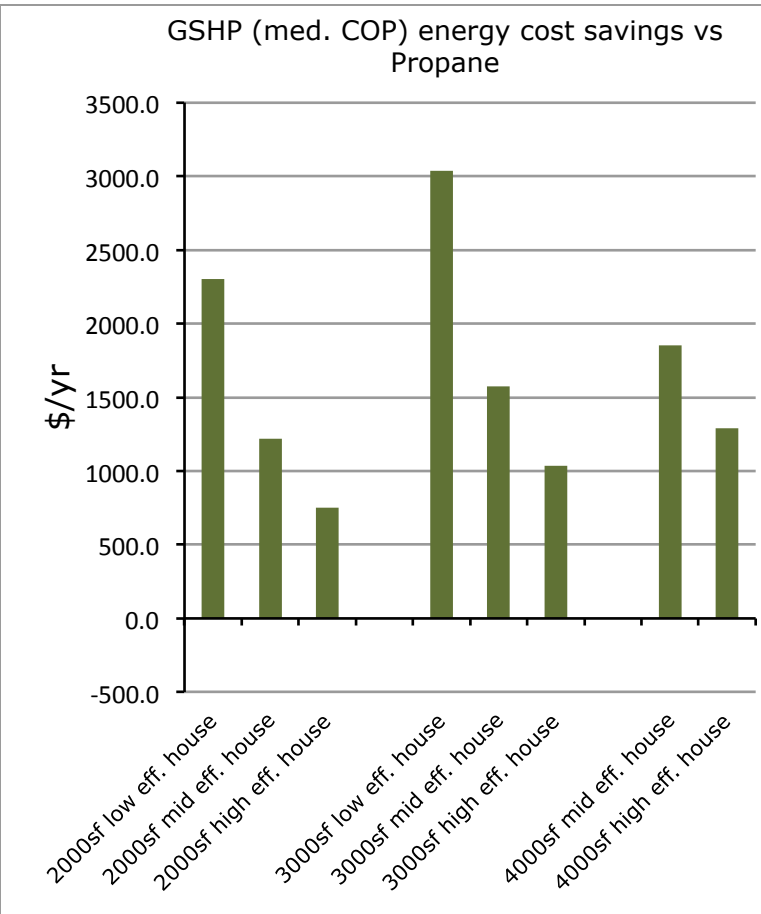
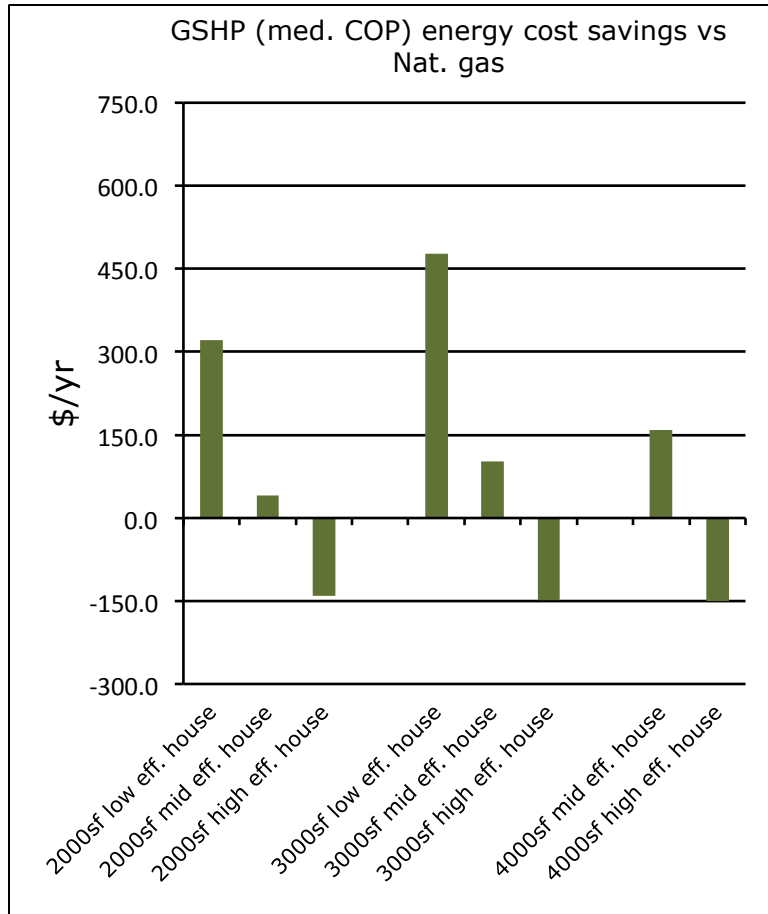
Rate assumptions for base case GFA systems (natural gas/propane)

Energy Type	Rate	Units	Time of Year	Source	\$/MMBtu
cooling energy charge	0.15	\$/kWh	June - Sep	2015 MN utility avg. (inc. surcharges)	\$43.96
heating fan energy charge	0.14	\$/kWh	Oct - May	2015 MN utility avg. (inc. surcharges)	\$41.03
nat. gas (water/space heat)	1.00	(\$/therm)	whole year	EIA 5yr avg for MN ('10-'15)	\$10.00
propane (water/space heat)	2.00	(\$/gallon)	whole year	EIA 5yr avg for MN ('10-'15)	\$21.90

Rate assumptions for GSHP systems (all electric)

Energy Type	Rate	Units	Time of Year	Source	\$/MMBtu
cooling energy charge	0.15	\$/kWh	June - Sep	2015 MN utility avg. (inc. surcharges)	\$43.96
heating energy charge	0.10	\$/kWh	Oct - May	2015 MN utility avg. (inc. surcharges)	\$29.31
water heating	0.117	\$/kWh	whole year	(weighted avg. of above)	\$34.19

GSHP - modeling



GSHP - modeling

Assuming a respectable level of performance (which might be a big if) GSHP can provide certain benefits in cold climates:

- 1) Reduced site energy use for all homes
- 2) Reduced site and source energy use for old, inefficient homes
- 3) For outstate areas, reduced site energy, source energy, CO2 emissions, and energy costs *compared to electric-resistance based systems*
- 4) For outstate areas, reduced and stabilized energy costs *for propane homes*

However, for most new homes in MN, GSHP systems increase source energy use, CO2 emissions, and energy costs.

In mixed-humid climates, results would be substantially different.

ASHP - background

Field research currently underway by Center for Energy and Environment:

*Cold Climate Air Source
Heat Pump Field Assessment*

Team members:

Nicole Kessler

Josh Quinnell

Ben Schoenbauer

Will install and monitor
performance of 6-8 air
source heat pumps in a variety
of installation conditions

www.mncee.org/heat_pumps

2 ductless models, 6 ducted

The screenshot shows the CEE website with a navigation bar at the top. The main content area features a large green-bordered box for '#WeAreCEE Staff Spotlights' with a sub-header and a paragraph of text. To the right of this box is a blue sidebar menu for 'Innovation Exchange' with links to Projects, Resource Center, Events & Webinars, About The Innovation Exchange, Staff, and Visit Our Blog. Below the main content are three columns: 'From The Blog' with a staff spotlight on Jennifer Amendt, 'News' with two articles, and 'Featured Staff' with profiles for Helen Booth-Tobin and Lloyd Dalton. The footer of the website is partially visible at the bottom of the screenshot.

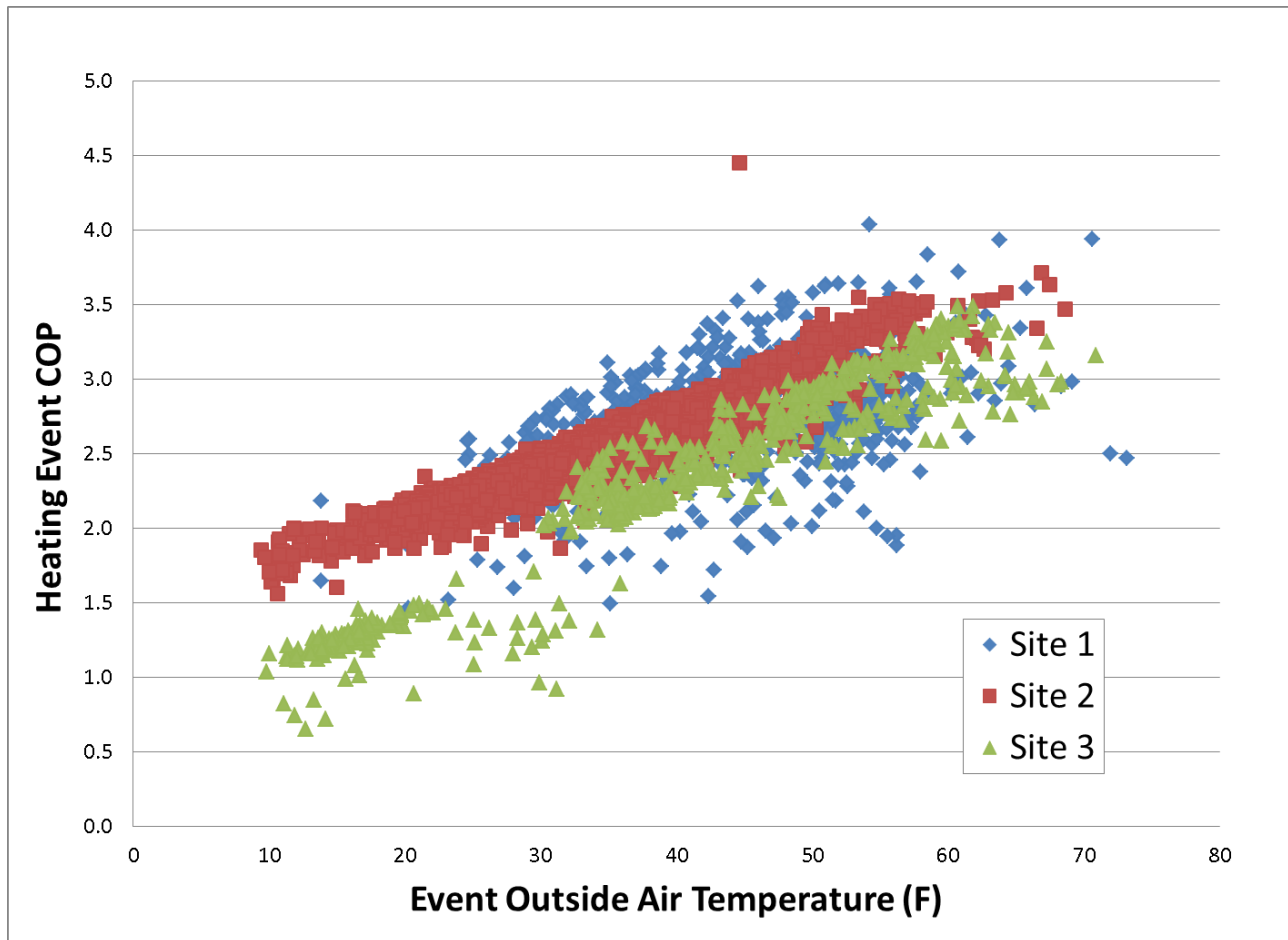
ASHP - background

Several complicating factors with ASHP efficiency:

1. COP varies considerably with temperature
2. In cold climates, COP impacted by backup heat
3. And defrost cycles

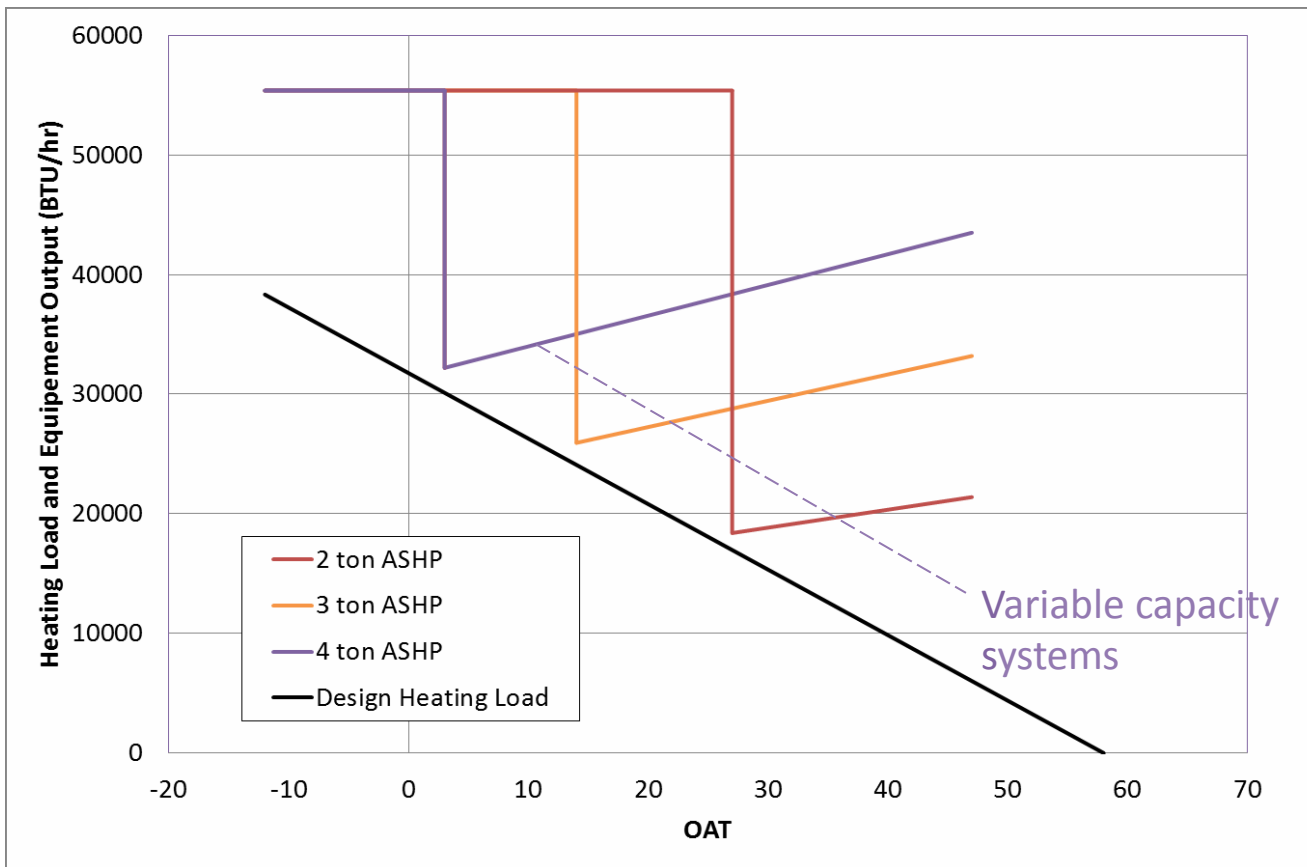
The screenshot shows the website for the Center for Energy and Environment (CEE). The header includes the CEE logo and navigation links: 'Contact Us | News', a search bar, and a 'Sign Up for Our Newsletter' button. The main navigation menu has several items: 'Who We Are', 'What We Do', 'Who We Work With', 'Our Impact', 'Innovation Exchange', and 'Find Financing & Incentives'. A dropdown menu for 'Innovation Exchange' is open, listing 'Projects', 'Resource Center', 'Events & Webinars', 'About The Innovation Exchange', 'Staff', and 'Visit Our Blog'. The main content area features a '#WeAreCEE Staff Spotlights' section with a text block and a 'Learn More' link. Below this are sections for 'From The Blog', 'News', and 'Featured Staff', each with a 'See All' link. The 'Featured Staff' section includes profiles for Helen Booth-Tobin and Lloyd Dalton.

ASHP Performance – Preliminary Data



- Rated COPs of 3.0-3.5 at 47 F
- COP observed
 - 1.5-3.5 (site 1 & 2)
 - 1-3.5 (site 3)
- 2015-2016 heating season
- Results shown with no backup, no frost-protection

Sizing



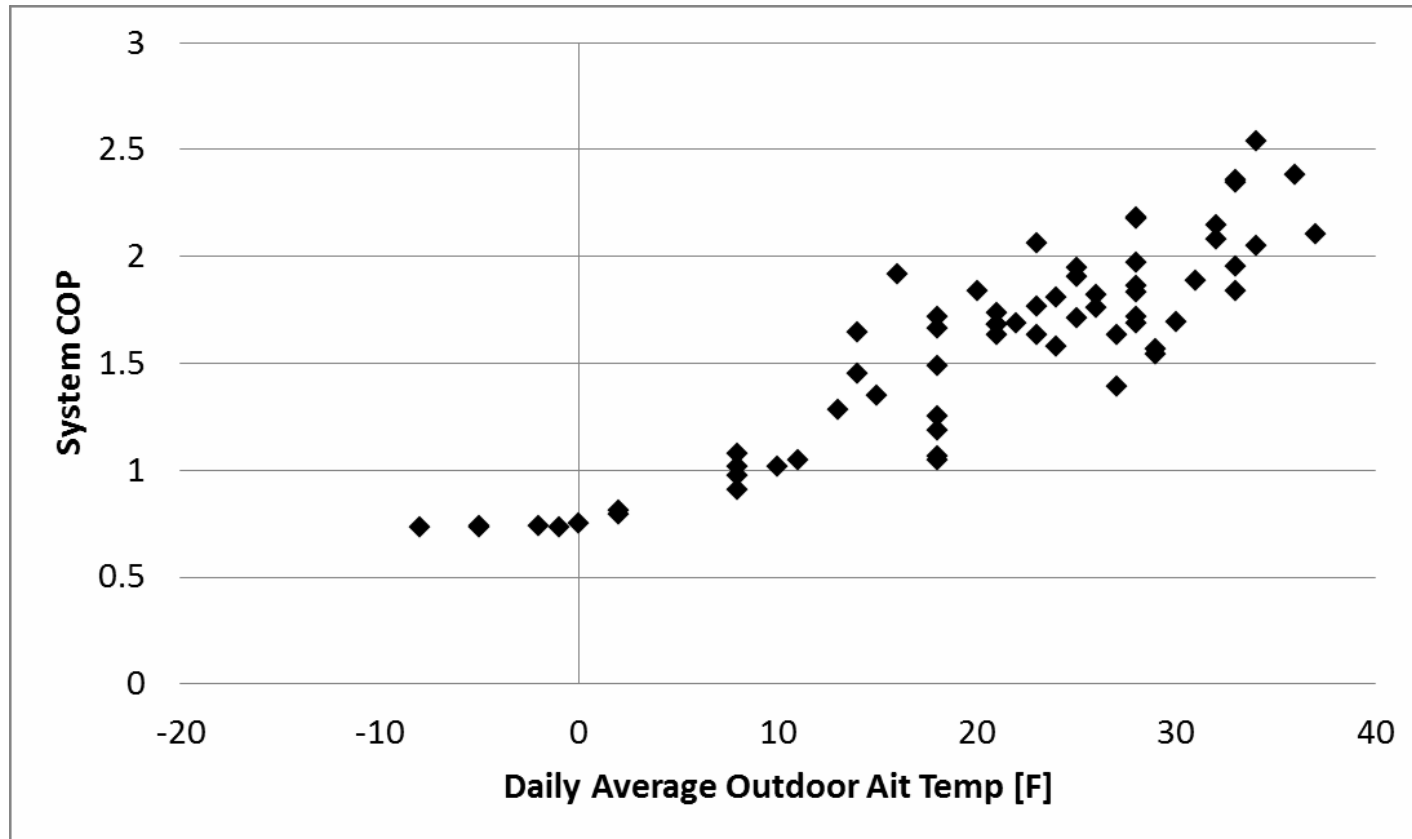
The OAT for the systems to switch to back up:

4ton ~3 F,
 3ton ~14 F,
 2 ton ~27F

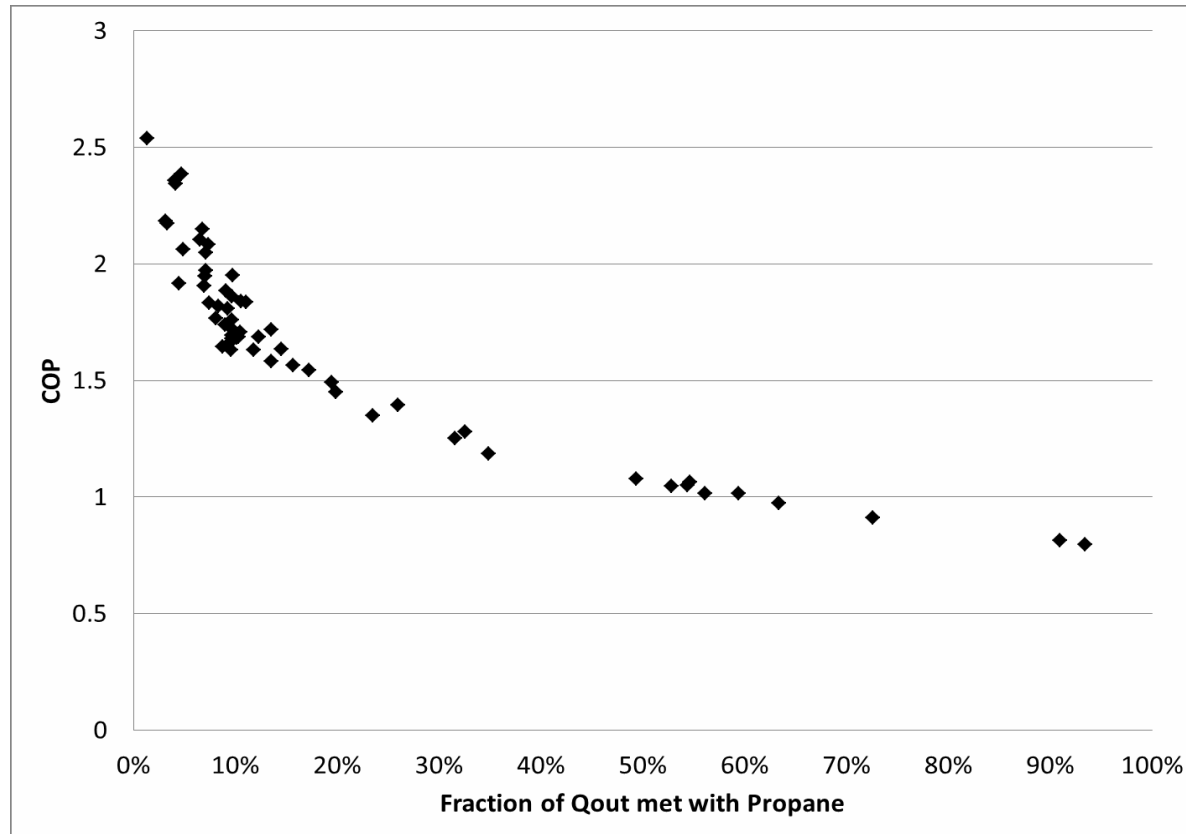
Percentage of heating load meet by ASHP:

4ton ~ 86%,
 3ton ~ 74%
 2 ton ~ 40%

Preliminary Results



Preliminary Results



Eventually, COP approaches furnace efficiency



Energy Use and Costs

- Assumptions:
 - 40,000 Btu/hr design heating load
 - Minneapolis TMY3
 - Equipment specifications

			LP Use	Electric Use	Annual Cost
	Avg COP	%ASHP	therms/yr	kWh/yr	\$/yr
Propane Furnace	0.80	0%	876	0	\$1,732
Propane Furnace	0.95	0%	735	0	\$1,453
ASHP w/30°F Change-over	1.18	27%	528	1,898	\$1,272
ASHP w/10°F Change-over	1.81	79%	152	6,859	\$1,124
ASHP w/0°F Change-over	1.97	89%	84	7,958	\$1,120



Energy Use and Costs

- Assumptions:
 - 40,000 Btu/hr design heating load
 - Minneapolis TMY3
 - Equipment specifications

			LP Use	Electric Use	Annual Cost	
	Avg COP	%ASHP	therms/yr	kWh/yr	\$/yr	ASHP COP:
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Propane Furnace	0.95	0%	735	0	\$1,453	
ASHP w/30°F Change-over	1.18	27%	528	1,898	\$1,272	3.42
ASHP w/10°F Change-over	1.81	79%	152	6,859	\$1,124	2.38
ASHP w/0°F Change-over	1.97	89%	84	7,958	\$1,120	2.27

ASHP w/0°F, electric backup	2.17	89%	0	2590 + 7958	\$1,266	2.27
-----------------------------	------	-----	---	-------------	---------	------

ASHP COP declines as it spends more time operating at cold temperatures, but...
 Avg. COP of total system goes up as backup use goes down.

Comparison and Discussion

How do GSHP and ASHP systems compare, and what are their implications for code houses and Passive Houses? Assume GFA-based code house is “base case”

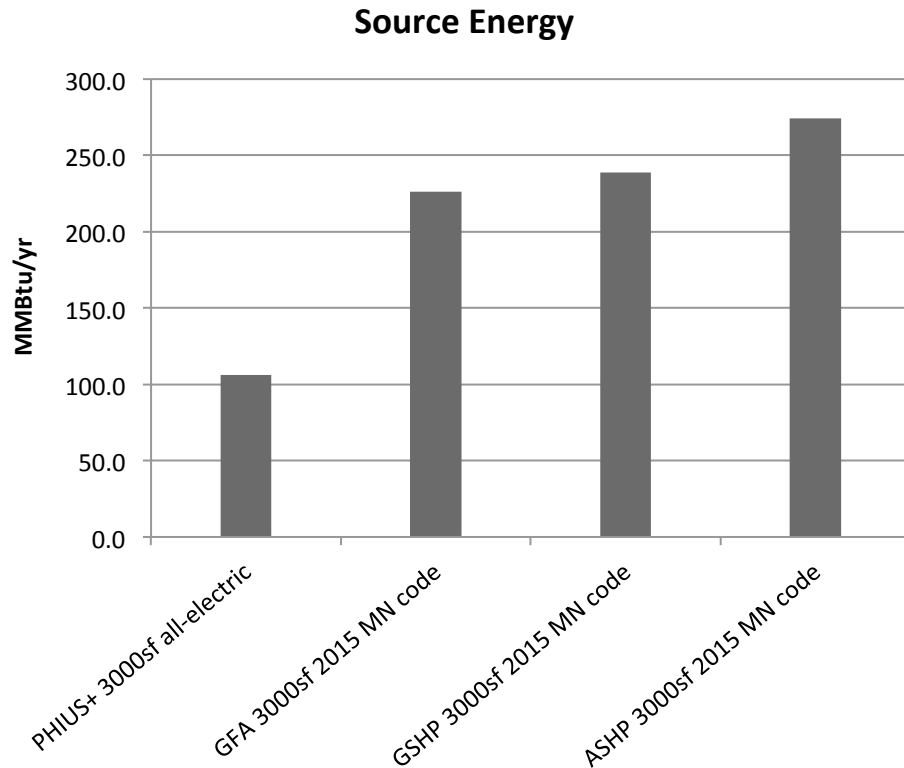
1. GSHP definitely provides highest efficiency
2. Both ASHP and GSHP will reduce *site* energy substantially

	heating	cooling
	COP	COP
GFA (base case)	0.90	3.4
GSHP (mid efficiency)	3.2	5.2
ASHP (ducted, w resistance backup and 0°F switchover)	2.2	4

Comparison and Discussion

How do GSHP and ASHP systems compare, and what are their implications for code houses and Passive Houses? Assume GFA-based code house is “base case”

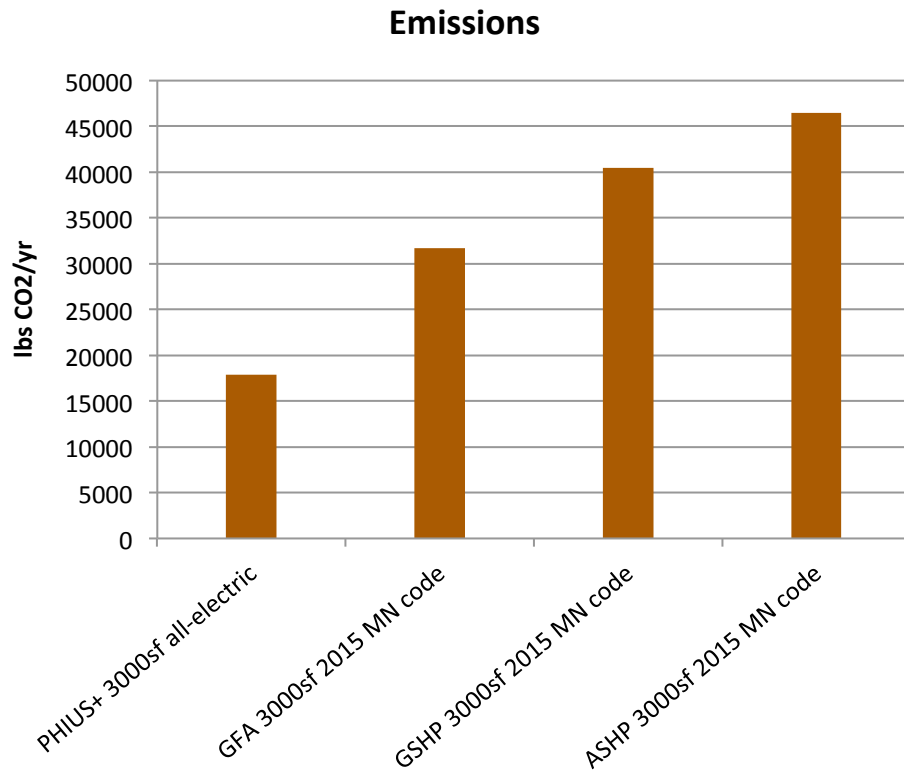
1. Compared to base case, GSHP and ASHP increase source energy slightly.
2. Compared to base case, PHIUS+ saves roughly 55% (6200 kWh x (bed + 1))



Comparison and Discussion

How do GSHP and ASHP systems compare, and what are their implications for code houses and Passive Houses? Assume GFA-based code house is “base case”

1. Compared to base case, GSHP and ASHP increase emissions significantly.
2. Compared to base case, PHIUS+ saves 45% (assuming all site energy is electric, which is worst case)



Implications for Passive House

We know the primary energy requirement is very tight. Maybe be the toughest requirement to hit without resorting to renewables. How do these systems stack up in terms of primary energy use?

Annual load x PE factor/efficiency = primary energy

Base case:

GFA system with 95AFUE, PE factor 1.09

$1.09/0.95 = \mathbf{1.15}$

This is the ratio to beat. A lower number saves primary energy.

Implications for Passive House

We know the primary energy requirement is very tight. Maybe be the toughest requirement to hit without resorting to renewables. How do these systems stack up in terms of primary energy use?

Annual load x PE factor/efficiency = primary energy

GSHP system:

Heating COP 3.2, PE factor 3.16

$3.16/3.2 = \mathbf{0.99}$

GSHP system will help reduce primary energy use for space heating.

Implications for Passive House

We know the primary energy requirement is very tight. Maybe be the toughest requirement to hit without resorting to renewables. How do these systems stack up in terms of primary energy use?

Annual load x PE factor/efficiency = primary energy

Ducted ASHP system:

Heating COP 2.2 (w 10% electric resistance backup), PE factor 3.16

$3.16/2.2 = \mathbf{1.44}$

ASHP system in very cold climates will increase primary energy use for space heating.

Implications for Passive House

We know the primary energy requirement is very tight. Maybe be the toughest requirement to hit without resorting to renewables. How do these systems stack up in terms of primary energy use?

Annual load x PE factor/efficiency = primary energy

Ducted ASHP system:

Heating COP 2.8 (w 10% electric resistance backup), PE factor 3.16

$3.16/2.8 = \mathbf{1.13}$

ASHP systems must achieve COP of 2.8 (including backup heating) to reduce primary energy use for space heating.

Implications for Passive House

We know the primary energy requirement is very tight. Maybe be the toughest requirement to hit without resorting to renewables. How do these systems stack up in terms of primary energy use?

Annual load x PE factor/efficiency = primary energy

Ducted ASHP system:

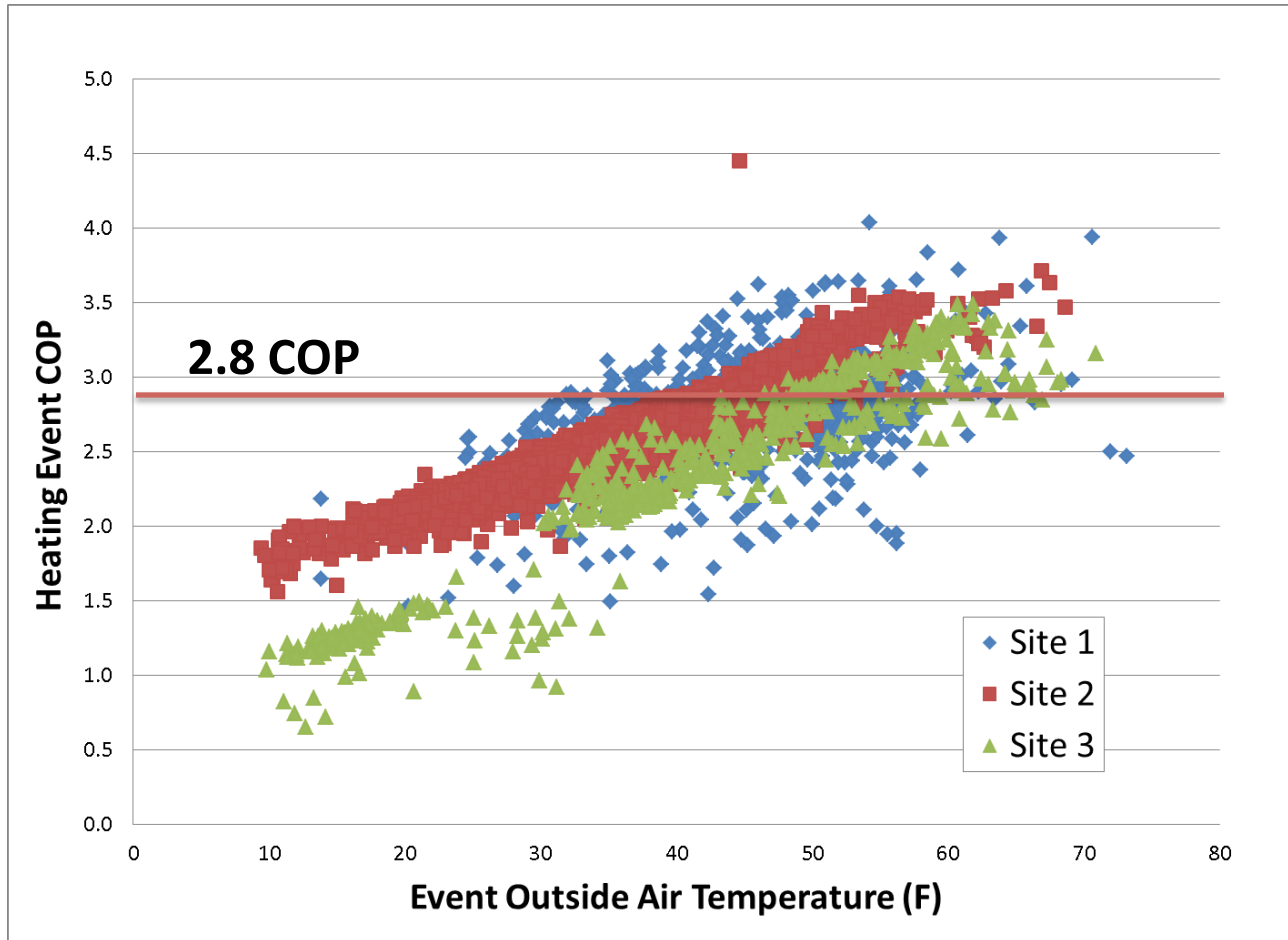
Heating COP 2.8 (w 10% electric resistance backup), PE factor 3.16

$3.16/2.8 = \mathbf{1.13}$

ASHP systems must achieve COP of 2.8 (including backup heating) to reduce primary energy compared to base case.

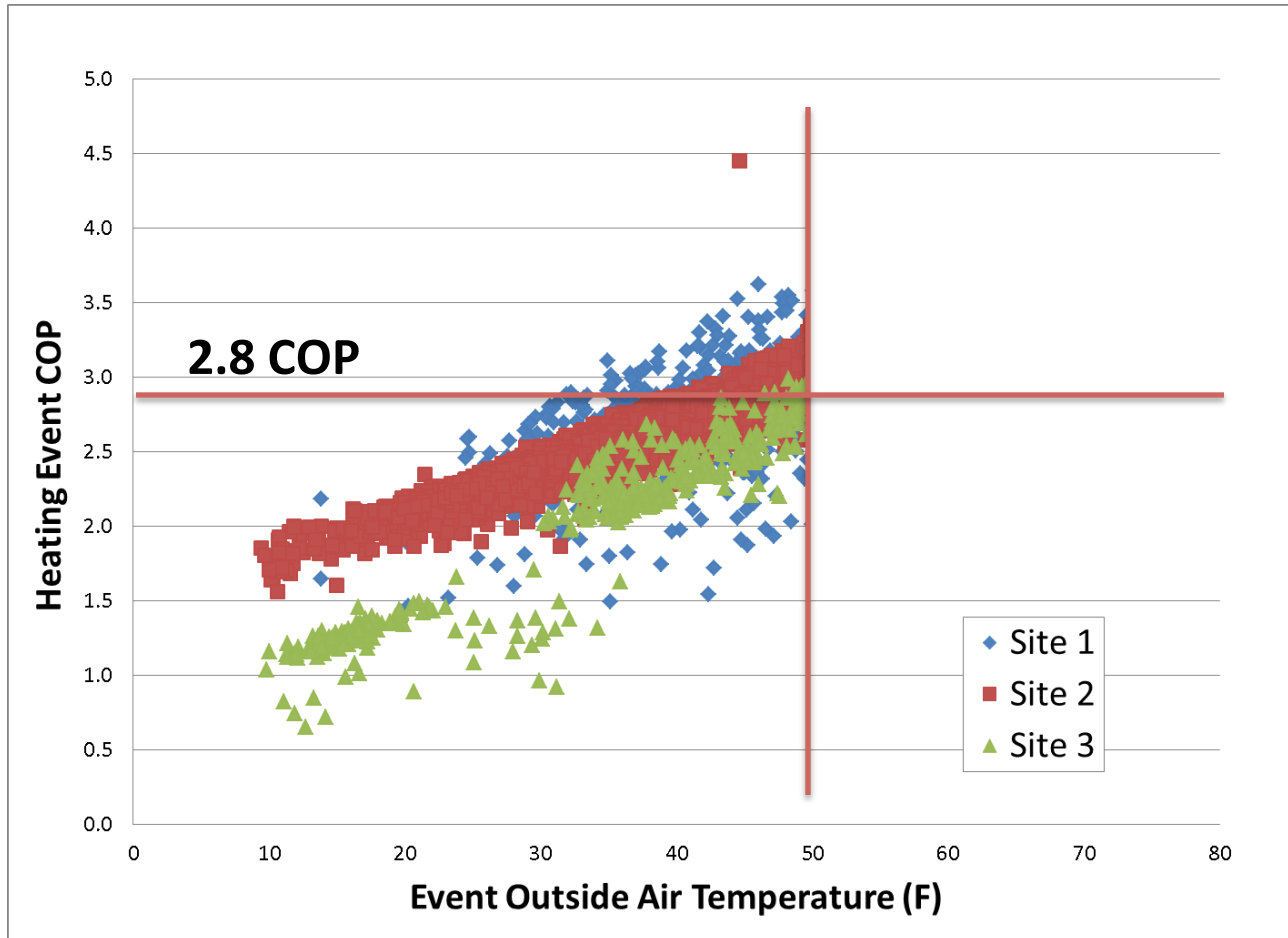
Data from CEE's monitoring on ducted cold climate ASHP suggests this may not be possible (yet) in Minnesota, or similar climate zone 6

ASHP Performance – Preliminary Data



- Rated COPs of 3.0-3.5 at 47 F
- COP observed
 - 1.5-3.5 (site 1 & 2)
 - 1-3.5 (site 3)
- 2015-2016 heating season
- Results shown with no backup, no frost-protection

ASHP Performance – Preliminary Data






- Rated COPs of 3.0-3.5 at 47 F
- COP observed
 - 1.5-3.5 (site 1 & 2)
 - 1-3.5 (site 3)
- 2015-2016 heating season
- Results shown with no backup, no frost-protection

And remember that with a Passive House, the balance point is shifted lower. The vast majority of heating load will be at temperatures below 50F

GSHP - modeling

3 home sizes

2000 sf house	3000 sf house	4000 sf house
<p data-bbox="195 462 291 486">26' x 26'</p> <p data-bbox="195 554 471 625">3-bed, 2-story with conditioned basement</p> <p data-bbox="195 689 658 761">0.15 window to floor area ratio 300 sf window area equally distributed</p>	<p data-bbox="730 462 871 486">31.5' x 31.5'</p> <p data-bbox="730 554 1006 625">4-bed, 2-story with conditioned basement</p> <p data-bbox="730 689 1193 761">0.15 window to floor area ratio 450 sf window area equally distributed</p>	<p data-bbox="1265 462 1406 486">36.5' x 36.5'</p> <p data-bbox="1265 554 1541 625">5-bed, 2-story with conditioned basement</p> <p data-bbox="1265 689 1729 761">0.15 window to floor area ratio 600 sf window area equally distributed</p>
		

GSHP - modeling

High efficiency (2015 MN Energy Code)	Values	Notes
Ceiling R-value	50.0	2015 MN Energy Code
Wall R-value	21.0	2015 MN Energy Code
Rim joist R-value	21.0	same as wall
Basement Wall R-value	15.0	2015 MN Energy Code
Slab R-value	10.0	2015 MN Energy Code
Fenestration U-factor (area-weighted avg.)	0.32	2015 MN Energy Code
Fenestration SHGC (area-weighted avg.)	0.26	approx. industry average
Airtightness (ACH@50Pa)	3.0	2015 MN Energy Code
Window-to-Floor Area Ratio	0.15	2012 IECC max
Furnace AFUE	90	2012 IECC
Air Conditioner SEER	13.0	2012 IECC
DHW ER	0.62	2012 IECC, 2015 NAECA
Ventilation Rate (cfm, continuous)	55/75/95	2015 MN Energy Code (2000/3000/4000sf)
Sensible Recovery Efficiency (%)	60.0	HRV (w separate kitchen fan)
Duct location		conditioned space
Lighting CFL percentage	80%	

3 home efficiency levels

“High efficiency” =
2015 MN energy code

GSHP - modeling

Med. efficiency (1980's MN Energy Code)	Values	Notes
Ceiling R-value	38.0	
Wall R-value	19.0	
Rim joist R-value	13.0	
Basement Wall R-value	10.0	
Slab R-value	10.0	
Fenestration U-factor (area-weighted avg.)	0.46	double clear vinyl (REM Rate library)
Fenestration SHGC (area-weighted avg.)	0.57	double clear vinyl (REM Rate library)
Airtightness (ACH@50Pa)	5,4,3	(est. rates 2000sf, 3000sf, 4000sf)
Window-to-Floor Area Ratio	0.15	
Furnace AFUE	80	
Air Conditioner SEER	10.0	
DHW ER	0.56	
Ventilation Rate (cfm, runtime)	50, 3/4/5hr	exhaust only (2000/3000/4000sf)
Sensible Recovery Efficiency (%)	0.0	no recovery
Duct location		conditioned space
Lighting CFL percentage	50%	

3 home efficiency levels

“Med. efficiency” =
1980’s MN energy code

GSHP - modeling

Low efficiency (Pre-Energy Code)	Values	Notes
Ceiling R-value	24.0	7 inches cellulose
Wall R-value	11.0	insulated 2x4 wall
Rim joist R-value	11.0	same as wall
Basement Wall R-value	5.0	1 inch interior foam board
Slab R-value	0.0	carpeted slab
Fenestration U-factor (area-weighted avg.)	0.67	single wood frame w storm (REM Rate lib.)
Fenestration SHGC (area-weighted avg.)	0.65	single wood frame w storm (REM Rate lib.)
Airtightness (ACH@50Pa)	7,6,5	(est. rates 2000sf, 3000sf, 4000sf)
Window-to-Floor Area Ratio	0.15	same as other models
Furnace AFUE	70	Gas Furn Pre 1987 (REM Rate library)
Air Conditioner SEER	9.0	Cent AC Pre 1987 (REM Rate library)
DHW ER	0.55	Gas Stor 1984-87 (REM Rate library)
Ventilation Rate (cfm, runtime)	50, 3/4/5hr	exhaust only (2000/3000/4000sf)
Sensible Recovery Efficiency (%)	0.0	no recovery
Duct location		conditioned space
Lighting CFL percentage	50%	

3 home efficiency levels

“Low efficiency” =
Pre-energy code with some
insulation retrofit